

Faculty of Civil and Environmental Engineering

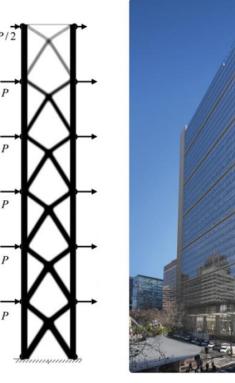
Optimal design of skeletal structures with buckling considerations using nonlinear beam modeling

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Structural optimization in civil engineering

• Examples from Skidmore, Owings & Merrill (SOM)

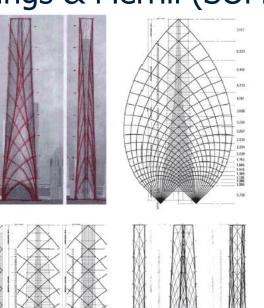


Beghini et al. 2013



Slide 2 out of 16

Truss optimization with buckling



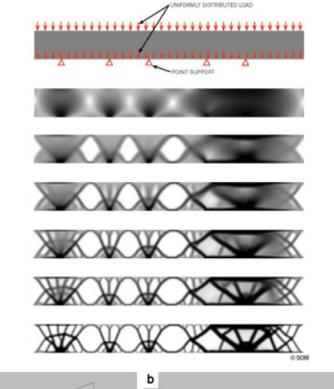


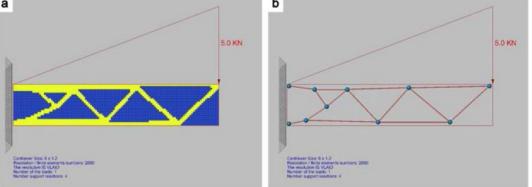
Besserud et al. 2013

Structural optimization in civil engineering

- Difficulties with interpreting complicated continuum forms
- Complicated conversion of continuum topology forms to practical construction







Mostafavi et al. 2013



Slide 3 out of 16

Besserud et al. 2013

Utilizing innovative trends

- Robotic construction of optimized trusses
- Optimization based on:
 - Ground structure approach
 - Classical plastic design (LP) formulation



"Practical" design



Design utilizing robotic capabilities





Fig. 57 robotic placing of bar member

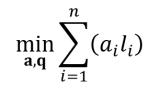




Slide 4 out of 16

Truss optimization using ground structure approach

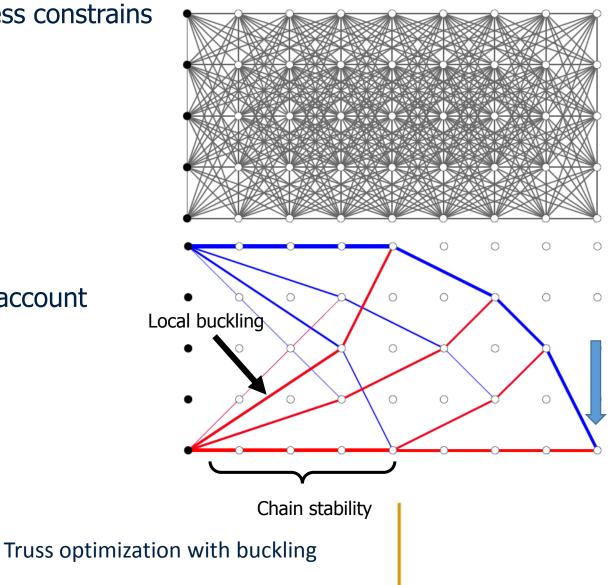
• Minimum weight / volume under stress constrains



s.t.:
$$\mathbf{Bq} = \mathbf{f}$$

 $a_i \sigma^{min} \le q_i \le a_i \sigma^{max}$
 $a_i \ge 0$, $i = 1..n$

- Classical approach doesn't take into account buckling considerations:
 - Local buckling
 - Global buckling
 - Chain stability



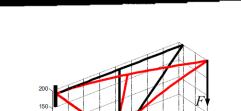


How are buckling considerations imposed in literature?

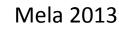
Achtziger 1999

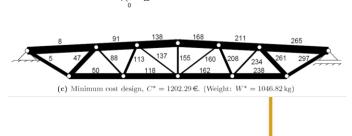


Ben-Tal et al. 2000, Kocvara et al. 2002



Torii et. al. 2014





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Slide 6 out of 16

- Plastic design problem sequentially
 - Euler buckling (constraint on each bar)
 - Chain stability (constraint on each sequence of unbraced connected bars)
- Semi-definite problem
 - Global buckling (one stability constraint)
 - Chain stability (by overlapping bars)
- Eigen-value problem
 - Global buckling
 - Local stability (constraints)
- Mixed-integer LP
 - Local buckling (constraint)
 - Chain stability (overlapping bars)

The aim of the current work

Account for all buckling considerations in a single formulation

Main idea

- Use geometric nonlinear (GNL) beam formulation
- Optimize the response of the nonlinear structure, instead of imposing constraints



Slide 7 out of 16

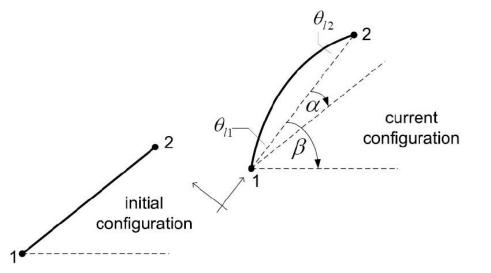
Geometric non-linear analysis

- GNL beam element derived using co-rotational formulation
- Kinematic assumptions:
 - Large displacements
 - Large rotations
 - Small strains

$$\mathbf{K}_{t} = \mathbf{B}^{T}\mathbf{C}\mathbf{B} + \frac{N}{l_{n}}\mathbf{z}\mathbf{z}^{T} + \frac{(M_{1} + M_{2})}{l_{n}^{2}}(\mathbf{r}\mathbf{z}^{T} + \mathbf{z}\mathbf{r}^{T})$$

- Computational scheme: Newton-Raphson
 - Displacement control equilibrium

$$\mathbf{R}(\mathbf{u},\theta) = \theta \hat{\mathbf{f}}_{\text{ext}} - \mathbf{f}_{\text{int}}(\mathbf{u}) = \mathbf{0}$$





Slide 8 out of 16

Problem formulation - maxF

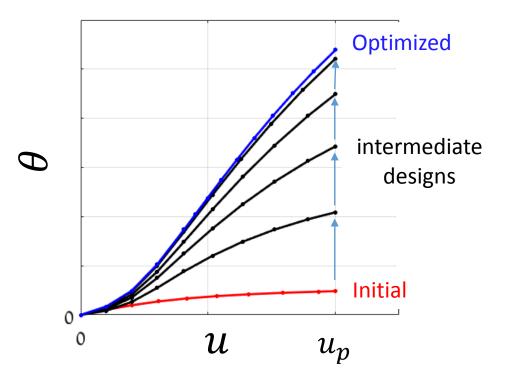
 Maximization of load-bearing capacity subject to a volume constraint with displacement control

$$\min_{\rho} f = -\theta$$

s.t.: $g = \sum_{i=1}^{n} (\rho_i a l_i) \le V^*$

$$\rho_{min} \le \rho_i \le \rho_{max} \quad i = 1..n$$

with:
$$\mathbf{R}(\rho, \mathbf{u}, \lambda) = \theta \hat{\mathbf{f}}_{ext} - \mathbf{f}_{int}(\rho, \mathbf{u}) = \mathbf{0}$$





Slide 9 out of 16

Problem formulation - minV

 Minimization of volume subject to a constraint on load-bearing capacity with displacement control

$$\min_{\rho} f = V$$

s.t.: $\mathbf{g} = \theta \ge \theta^*$
 $\rho_{min} \le \rho_i \le \rho_{max}$ $i = 1..n$
with: $\mathbf{R}(\rho, \mathbf{u}, \lambda) = \theta \hat{\mathbf{f}}_{ext} - \mathbf{f}_{int}(\rho, \mathbf{u}) = \mathbf{0}$
 θ^*
 θ^*

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Slide 10 out of 16

Sensitivity analysis and solution

- Non-linear programming by the Method of Moving Asymptotes (MMA) gradient based algorithm
- Sensitivity analysis following the adjoint method:

$$\hat{c}(\theta, \mathbf{\rho}, \mathbf{u}) = c(\theta, \mathbf{\rho}, \mathbf{u}) - \boldsymbol{\lambda}^T [\theta \mathbf{f}_{ext} - \mathbf{f}_{int}(\mathbf{\rho}, \mathbf{u})]$$

$$\frac{\partial \hat{c}}{\partial \rho_{e}} = \left(\frac{\partial c}{\partial \mathbf{u}} + \lambda^{T} \frac{\partial \mathbf{f_{int}}}{\partial \mathbf{u}}\right) \frac{\partial \mathbf{u}}{\partial \rho_{e}} + \left(\frac{\partial c}{\partial \theta} - \lambda^{T} \mathbf{f_{ext}}\right) \frac{\partial \theta}{\partial \rho_{e}} + \lambda^{T} \frac{\partial \mathbf{f_{int}}}{\partial \rho_{e}}$$
$$\frac{\partial \hat{c}}{\partial \rho_{e}} = \lambda^{T} \frac{\partial \mathbf{f_{int}}}{\partial \rho_{e}} \qquad \begin{bmatrix} K_{ff} & K_{fp} \\ f_{ext}^{f} & f_{ext}^{p} \end{bmatrix} \begin{bmatrix} \lambda_{f} \\ \lambda_{p} \end{bmatrix} = \begin{bmatrix} -\frac{\partial c}{\partial u_{f}} \\ \frac{\partial c}{\partial \theta} \end{bmatrix}$$

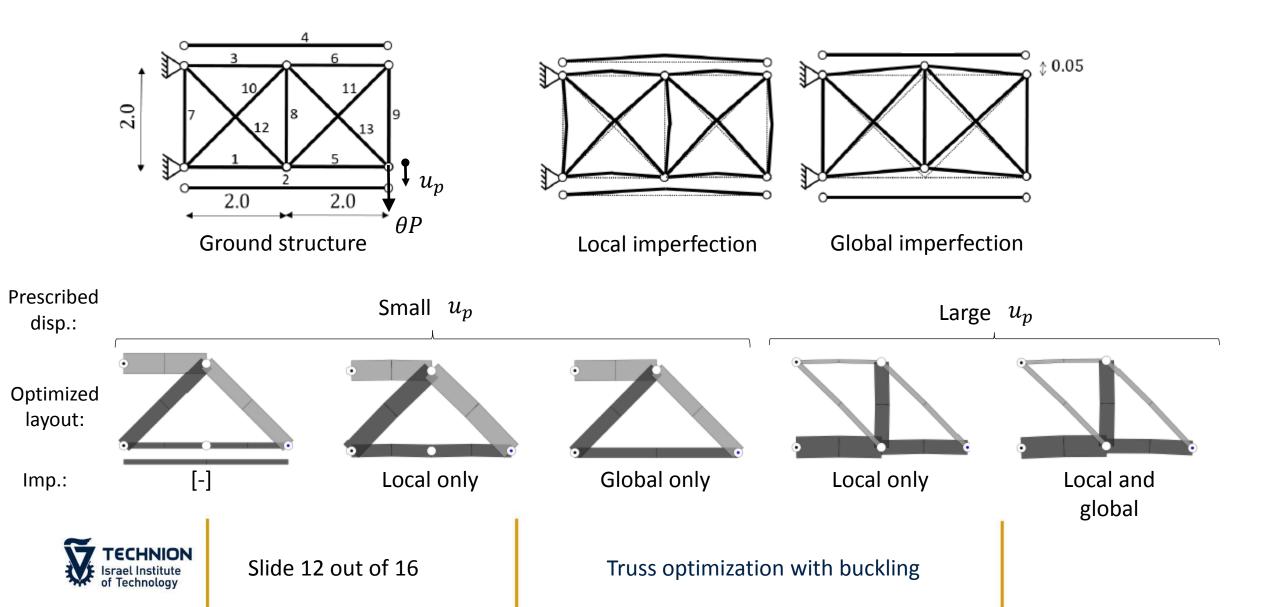
$$c(\theta, \mathbf{\rho}, \mathbf{u}) = -\theta$$

$$\begin{bmatrix} K_{ff} & K_{fp} \\ f_{ext}^f & f_{ext}^p \end{bmatrix} \begin{bmatrix} \lambda_f \\ \lambda_p \end{bmatrix} = \begin{bmatrix} 0 \\ -1 \end{bmatrix}$$



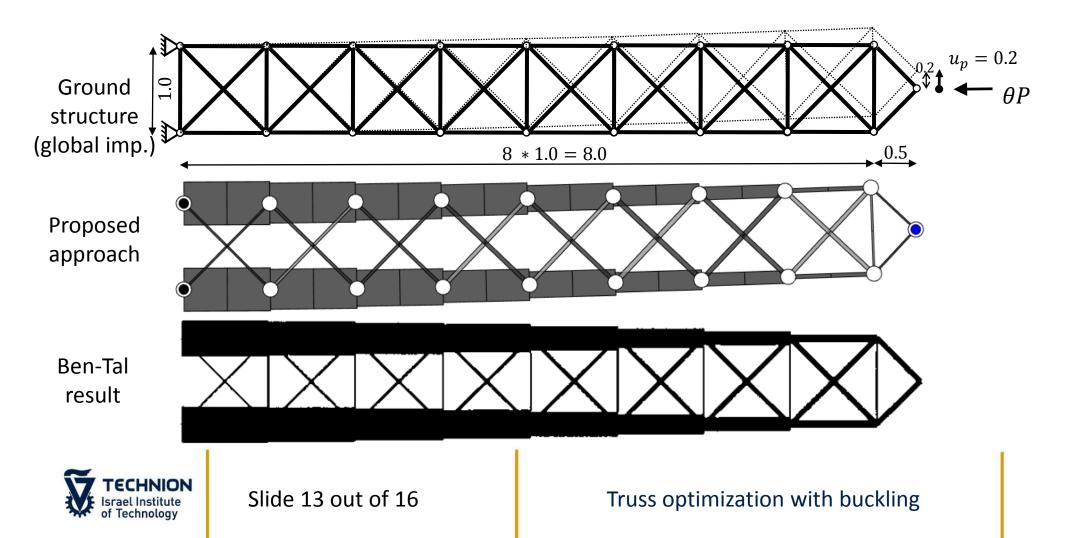
Slide 11 out of 16

Demonstration: the effect of imperfections



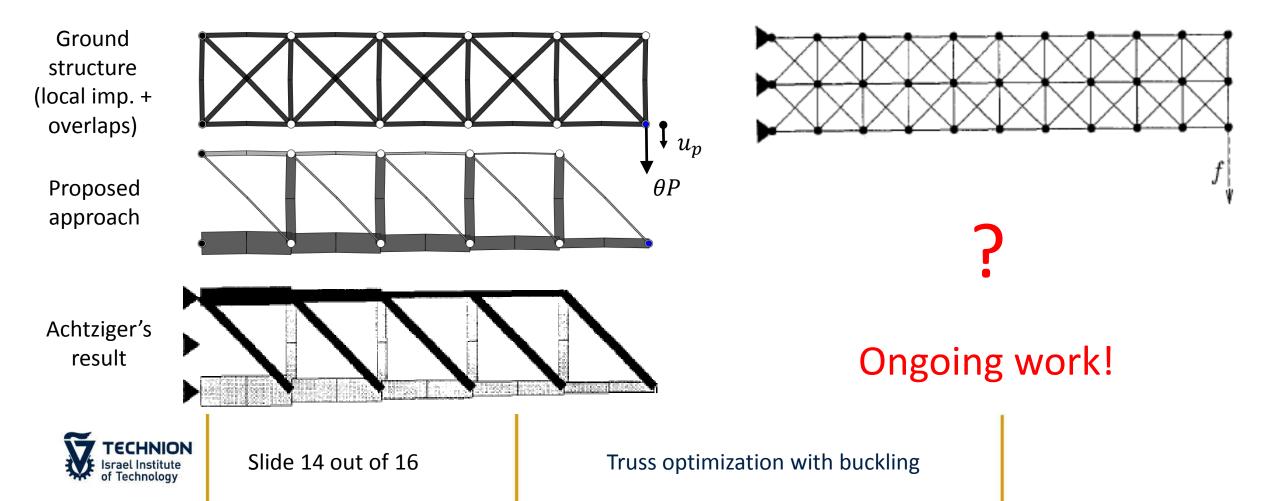
Preliminary results

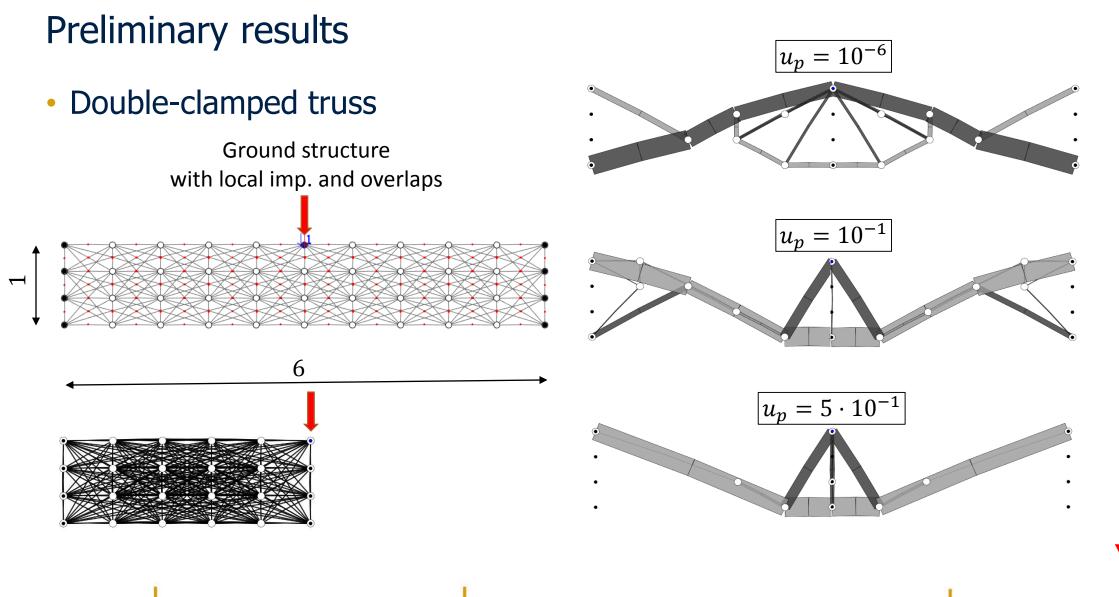
• Global buckling of a cantilever – Ben-Tal et al. 2000



Preliminary results

Local stability of a cantilever – Achtziger 1999





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Slide 15 out of 16

Summary

- GNL beam model has been integrated into truss optimization
- Considering buckling through the nonlinear response can replace the imposition of buckling constraints
- Preliminary results resemble those achieved by more traditional approaches involving constraints on local and global stability

Ongoing work

- Choice of local eccentricity Euler buckling
- Choice of prescribed displacement
- Choice of global imperfection initial positions of nodes
- Dealing with convergence difficulties for intermediate designs



Thank you!

